

GUJARAT TECHNOLOGICAL UNIVERSITY**BE - SEMESTER-IV • EXAMINATION – SUMMER 2013****Subject Code: 140605****Date: 17-06-2013****Subject Name: Advanced Strength of Materials****Time: 10.30 am - 01.00 pm****Total Marks: 70****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.

- Q.1** (a) Derive expressions for finding out strain energy stored in a member due to (i) Impact load (ii) tension **04**
- (b) State Castigliano's first theorem and Maxwell's reciprocal theorem. **03**
- (c) A uniform metal bar of rectangular section 40mm × 40mm is of length 1.5m. Find the Strain energy stored in the bar when a load of 120 kN is gradually applied to it. If the elastic limit of the metal with which the bar is made is 160 N/mm², also find proof resilience and modulus of resilience. **07**
- Q.2** (a) Stating assumptions derive Lamé's equations to find out the stresses in a thick cylindrical shell. **07**
- (b) A hollow cylinder has an external diameter of 250 mm and thickness of the wall is 25mm. The cylinder is subjected to an internal fluid pressure = 35MPa and external pressure = 3.5 MPa. Calculate the maximum and minimum circumferential stresses and plot the variation of the same across the wall thickness. **07**

OR

- (b) A compound cylinder is made by shrinking a cylinder of external diameter 200 mm and internal diameter 160mm over another cylinder of external diameter 160mm and internal diameter 120mm. The radial pressure at junction after shrinking is 9 MPa. Find the final stress set up across the section, when compound cylinder is subjected to an internal fluid pressure of 70 MPa. **07**
- Q.3** (a) Explain given failure theories (i) Maximum Principal strain theory (ii) Maximum strain energy theory. **07**
- (b) A member having square cross section is subjected to axial pull of 12 kN and shear force of 5 kN. Design the cross section of member based on (i) The maximum principal stress theory (ii) The maximum shear stress theory. For a member elastic limit in axial tension is 250 MPa, Poisson's Ratio = 0.3 and Factor of safety = 2.5. **07**

OR

- Q.3** (a) Find an expression for the bending moment in a circular ring which is subjected to a tensile load along the diameter. **07**
- (b) A curved beam, rectangular in cross-section is subjected to pure bending of 600 Nm. The beam has width of 30mm, depth of 40mm and curved in a plane parallel to the depth. The mean radius of curvature is 50mm. Find the position of neutral axis, maximum bending compressive stress and maximum bending tensile stress. Also plot the variation of the bending stress across the section. **07**

- Q.4 (a)** Derive the equation of shear stress, bending stress, deflection and angular rotation for open helical spring. **07**
- (b)** A laminated steel spring simply supported at ends with span of 0.8 m is centrally loaded with a load of 8 kN. The central deflection under the above load is not to exceed 50mm and the maximum stress is to be 400 MPa, determine; (i) width of plate (ii) thickness of plate (iii) number of plates (iv) the radius to which plates should be bent so that the spring become straight under the given 7.5 kN load. Assume width= 12 x thickness and E= 210 GPa. **07**

OR

- Q.4 (a)** A central horizontal section of a hook is a symmetrical trapezium 90mm deep. The inner width being 100mm and outer being 50mm. the hook carries a load of 70 kN, the load line passes at a distance of 40mm from the inside edge of the section. The centre of curvature is in the load line. Calculate the extreme intensities of stresses. Also plot the stress distribution across the section. **07**
- Q.4 (b)** A flat spiral spring is 6mm wide and 0.25mm thick, the length is 3m. Assuming the maximum stress of 800 MPa to occur at the point of greatest bending moment, determine (i) The torque (ii) The work stored in the spring (iii) The number of turns to wind up spring. Take E=210 GPa. **07**
- Q.5 (a)** Plot shear stress distribution diagram for any three standard sections. **03**
- (b)** A simply supported timber beam of rectangular cross-section 120mm wide and 250mm deep. The span of beam is 5m and permissible shear stress is 5MPa. Compute the allowable UDL on beam. **04**
- (c)** A thin channel section has outside flange and web dimensions of 102mm and 204mm respectively. The thickness of flanges and web is uniform and equal to 4mm. Draw the shear stress and shear flow distribution for the section and find the position of the shear center. Take value of shear force at section = 50 kN. **07**

OR

- Q.5 (a)** Derive expression of hoop stress in rotating ring (rim). **03**
- (b)** A flywheel rim of 6m mean diameter is rotated so that maximum hoop stress in the material is 8 MPa. If density of the material of fly wheel is 7000 kg/m³, find the allowable speed neglecting arm effect. **04**
- (c)** A flat steel disc of uniform thickness and 1m diameter rotates at 3600 r.p.m. Determine the intensities of principal stresses. Take density of material is 7.85×10^{-5} N/mm³ and Poisson's Ratio = 1/3. **07**

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