

GUJARAT TECHNOLOGICAL UNIVERSITY**M. E. - SEMESTER – I • EXAMINATION – SUMMER • 2013****Subject code: 711503****Date: 13-06-2013****Subject Name: Advanced Solid Mechanics****Time: 10.30 am – 01.00 pm****Total Marks: 70****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.

Q.1	(a)	State the types of failure associated with structure and show the various stability criteria for generating alternative configuration.	07
	(b)	Explain equilibrium approach and derive its general equation to get critical load for end condition as one end roller and one end fixed.	07
Q.2	(a)	Show that the following 2-D state of stresses without body forces is in equilibrium: $\sigma_x = 3x^2 + 4xy + 8y^2$ $\sigma_y = 2x^2 + xy + 3y^2$ $\tau_{xy} = \frac{1}{2}x^2 + 6xy + 2y^2$	04
	(b)	For the following state of stresses, find the principal stresses and the direction cosines of any ONE principal stress. Normal stresses: $\sigma_{xx} = 400$ MPa, $\sigma_{yy} = 100$ MPa, $\sigma_{zz} = 650$ MPa, and Shear stresses: $\tau_{xy} = 60$ MPa, $\tau_{yz} = 620$ MPa, $\tau_{zx} = 650$ MPa,	10
		OR	
	(b)	Derive the equation $\nabla^4 \phi = 0$ for polar co-ordinate system.	10
Q.3	(a)	Discuss energy approach for stability of column and derives its general equation to get critical load for end condition as one fixed and one end free.	07
	(b)	Discuss imperfection approach and state the principle of imperfection for stability of column. Derive general equation of deflection to study initial effect of curvature.	07
		OR	
Q.3	(a)	Using trigonometry series determine the value of critical load by assuming suitable shape of curve. State the advantages of energy approach.	07
	(b)	Derive the standard equation for buckling of frames to get critical load. Use anti-symmetrical buckling.	07
Q.4	(a)	Drawing neat sketch for a curved beam subjected to bending moment. Also, give various boundary conditions for the same.	04
	(b)	For the curved beam subjected to moment: $M = 300$ kJ, internal & external radii: $a = 360$ mm & $b = 200$ mm respectively, calculate radial and transverse stresses at inner, outer and every quarter thickness points and plot their variations using the following equations with usual notations:	10

		Radial stress: $\sigma_r = \frac{4M}{N} [a^2 b^2 / r^2 \ln(b/a) + b^2 \ln(r/b) + a^2 \ln(a/r)]$ Tangential stress: $\sigma_\theta = \frac{4M}{N} [a^2 b^2 / r^2 \ln(b/a) + b^2 \ln(r/b) + a^2 \ln(a/r) + b^2 - a^2]$ Here; $N = (b^2 - a^2)^2 - 4 a^2 b^2 [\ln(b/a)]^2$	
		OR	
Q.4	(a)	Derive strain displacement relation in Cartesian co-ordinate system.	07
	(b)	Derive the following relation with usual notations : $\frac{1}{2}(\sigma_x + \sigma_y) + \frac{1}{2}(\sigma_x - \sigma_y) \cos 2\theta = \sigma_{xy} \sin 2\theta$	07
Q.5	(a)	Using Swift construction, find normal and resultant shear stress on a plane whose normal has directions cosines are $l = 0.342$, $m = 0.544$ and $n = 0.766$ respectively w.r.t. Principal stresses: $P_1 = 700$ MPa (Tensile), $P_2 = 250$ MPa (Tensile) and $P_3 = 300$ MPa (Compressive).	07
	(b)	Using beam column theory derive basic equation for a beam of length l resting on two simple supports carrying concentrated lateral load Q at centre of the beam.	07
		OR	
Q.5	(a)	Drawing neat sketch, explain the soap-bubble analogy of torsion in and derive the equation $\phi = (2 C \theta S/p) z$ with usual notations.	07
	(b)	State the differential equation for the case of non-conservative forces for column with one end fixed and one end free condition using dynamic criteria of stability.	07