

GUJARAT TECHNOLOGICAL UNIVERSITY
M. E. - SEMESTER – I • EXAMINATION – SUMMER • 2014

Subject code: 711503N**Date: 19-06-2014****Subject Name: Advanced Solid Mechanics****Time: 02:30 pm - 05:00 pm****Total Marks: 70****Instructions:**

1. Attempt all questions.
2. Make suitable assumptions wherever necessary.
3. Figures to the right indicate full marks.

- Q.1** (a) Explain Octahedral Planes and derive the expressions for normal stresses and shear stresses for such planes. **07**
- (b) Elaborate soap film bubble analogy and derive the following equation with usual notations: $\sqrt{2} \left(\frac{1}{2} G \right) = \sqrt{2} \left(\frac{s_z}{p} \right) = \phi_1$. **07**
- Q.2** (a) Draw the neat sketch for an element subjected to body forces, radial stresses, transverse stresses and shear stresses. Also, explain the basic differential equations of equilibrium in 2-D Polar Coordinate System. (no need for derivation) **07**
- (b) Draw the neat sketch for the displacement of an element. Also, explain the equation for various strains in Polar Coordinate System with its derivation. **07**
- OR**
- (b) Explain the generalized Hooke's law. Derive the stress-strain relationship for Isotropic materials and the relationship between the elastic constants. **07**
- Q.3** (a) Derive the equation of curved beams subjected to bending moment. Also, give various boundary conditions for the same. **07**
- (b) Find the principal stresses using CARDAN's method and the direction cosines of principal stresses. **07**
- Normal stresses: $\sigma_{xx} = 700 \text{ MPa}$, $\sigma_{yy} = 150 \text{ MPa}$, $\sigma_{zz} = 120 \text{ MPa}$, and
 Shear stresses: $\tau_{xy} = 25 \text{ MPa}$, $\tau_{yz} = 40 \text{ MPa}$, $\tau_{zx} = 30 \text{ MPa}$,
- OR**
- Q.3** (a) For a solid circular plate of small uniform thickness, material density ρ , rotating about the center with angular velocity ω , derive the expression for radial and tangential stress. **07**
- (b) Find the principal stresses and direction cosines of any one principal stress for the following state of stresses. **07**
- $$\begin{bmatrix} 9 & 6 & 3 \\ 6 & 5 & 2 \\ 3 & 2 & 4 \end{bmatrix} \text{ MPa}$$
- Q.4** (a) What is the concept of stability of structures? Give basis of stability of analysis for a slender straight column as well as column initially bent. **07**
- (b) A solid circular shaft up of Cast Iron is 2.2 m long & fixed at one end is subjected to a torque $T = 700 \text{ N-m}$ at the free end. Determine the smallest radius so that it does not fail according to the maximum normal stress theory. The ultimate tensile stress of CI is 160 MPa. **07**
- OR**
- Q.4** (a) Discuss effect of transverse shear on buckling of the beam & derive equation of critical load for the same. **07**
- (b) Derive equation of buckling load & deformation for the column with one end free & other fixed which produces structural instability. **07**

- Q.5 (a)** At a point in a strained ductile material, the biaxial state of stresses are acting as $\sigma_x = -150$ MPa (C), $\sigma_y = 0$ & $\tau_{xy} = 150$ MPa. If yield strength of the material is 260 MPa, check whether the material is safe using maximum shear stress theory and/or maximum distortion energy theory. **07**
- (b)** Give characteristics of Airy's stress function. Is $\phi = A (y^4 - 6x^2y^2)$ representing Airy's stress function? Here, A is a constant. **07**

OR

- Q.5 (a)** Derive the following equation with usual notations: **07**

$$\sigma_x = \frac{1}{2} (\sigma_x + \sigma_y) + \frac{1}{2} (\sigma_x - \sigma_y) \cos 2\theta + \tau_{xy} \sin 2\theta$$
- (b)** Given the following stress field in MPa : **07**
 $\sigma_x = 1.38x^3 + 0.0552y$, $\sigma_y = 1.104x^2 + 13.8$ & $\tau_{xy} = 2.76x + 2.208y^2$.
 Calculate strain at a point (3, 5, 2). Assume $E = 220$ GPa & Poisson's ratio $\nu = 0.18$.
