Seat No.:	
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Enrolment

## GUJARAT TECHNOLOGICAL UNIVERSITY PDDC SEMESTER- VI • EXAMINATION – SUMMER- 2015

Subject Code :X61903 Subject Name: Heat & Mass Transfer. Time:10:30 am - 01:00 pm Instructions: Date:14 /05/2015

**Total Marks: 70** 

- 1. Attempt all questions.
- 2. Make suitable assumptions wherever necessary.
- 3. Figures to the right indicate full marks.
- Q.1 (a) With usual notations derive general heat conduction equation in Cartesian 07 coordinates. Also obtain the same for no heat sources case.
  - (b) Distinguish between the Conduction, Convection, & radiation mode of heat transfer.
- Q.2 (a) Derive equation of temperature distribution & heat dissipation for infinite long fin. 07
  - (b) A wall of a furnace is made up of inside layer of silica brick 120 mm thick covered with a layer of magnesite brick 240 mm thick. The temperature at the inside surface of silica brick wall & outside surface of magnesite brick wall are 725° C & 110° C respectively. The contact thermal resistance between the two walls at interface is 0.0035° C/W per unit wall area. If the thermal conductivities of silica & magnesite bricks are 1.7 W / mK & 5.8 W / mK,

Calculate :

(1) Rate of heat loss per unit area of walls & (2) Temperature drop at the interface.

## OR

(b) A furnace wall, 32 cm thick, is made up of an inner layer of brick ( k = 0.84 W/mk ) covered with a layer of insulation ( k = 0.16 W/m<sup>0</sup> K ). The furnace operates at a temperature of 1325<sup>o</sup> C & the ambient temperature is 25<sup>o</sup> C.
(i) Determine the thickness of brick & insulation which gives minimum heat loss.

(ii) Calculate the heat loss presuming that the insulating material has a maximum temperature of  $1200^{\circ}$  C. If the calculated heat loss is not accepted than state whether addition of another layer of insulation would provide a satisfactory solution.

- Q.3 (a) Derive expression for the logarithmic Mean Temperature Difference for the flow 07 in a counter flow heat exchanger.
  - (b) How does dropwise condensation differ from film condensation ? Which mode of condensation is characterized by lrger heat transfer rates ?

- **Q.3** (a) Describe the phenomenon of radiation from real surface.
  - (b) What is the StephenBoltzmann Law ? Explain the concept of total emissive power 07 of a surface.
- Q.4 (a) Distinguish (1) Subcooled & Saturated boiling. (2) Nucleate & film boiling. 07
  - (b) Derive an expression for LMTD for counter flow heat exchanger. 07

## OR

- Q.4 (a) By dimensional analysis show that for forced convection heat transfer the Nusselt 07 number can be expressed as a function of Prandlt number & Reynolds number.
  - (b) A copper pipe carrying referigent at -20° C is 10 mm in outer diameter & is exposed to convection at 50 W/m² K to air at 25°C. It is proposed tp apply insulation of conductivity 0.5 W/m K. Determine the thickness beyond which the heat gain will be reduced. Calculated the heat gains for 2.5mm, 5mm, 7.5mm & 20mm thickness for 1 m length. The convection coefficient remains same.
- Q.5 (a) Drive the expression for radiant heat exchange between two finite black surfaces 07 by radiation.
  - (b) Turbine blade of alloy steel ( K = 29 W/mK, 6 cm long,  $Acs = 500 \text{mm}^2 \& p = 07$ 12cm ) is exposed to hot gases of  $820^\circ$  C. The root temperature is  $480^\circ$  C. Determine the temperature in the middle of the blade & heat loss from the blade. Take h=320 W/m<sup>2</sup>K.

## OR

- Q.5 (a) Define & disuss velocity boundry layer & thermal boundry layer over a flat plate. 07 Show the thickness of these layers for different Prandtl numbers.
  - (b) The air at atmospheric pressure & temperature of 30°C flows over one side of the plate with a velocity of 90 m/min. The plate is heated & maintained at 100° C over its entire length, Find out the following at 0.3 m & 0.6 m from its leading edge.
    - (i) Thickness of velocity boundry layer & thermal boundry layer.
    - (ii) Mass flow rate which enters the boundry layer between 0.3 m & 0.6 m per metre depth of plate. Assume unit width of plate. Properties of air at  $30^{0}$  C are as follows :

$$\label{eq:prod} \begin{split} P &= 1.165 \ kg/m3, \ v = 16 x 10^{-6} \ m^2/s, \ Pr = 0.701, \\ Cp &= 1.005 \ kj/kg-K, \ k = 0.02675 \ W/m-K. \end{split}$$

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